

PART A / BAHAGIAN A

- (1). (a). Assess the importance of fatigue life and crack growth behaviour of rubber in predicting the lifetime of rubber engine mount products based on the S-N curve.

Nilaikan kepentingan kelakuan hayat lesu dan perambatan carikan getah dalam meramalkan jangka hayat bagi product cagak enjin berdasarkan lekuk S-N.

(10 marks/markah)

- (b). A bonded rubber unit has or present the following properties:

Satu unit getah terikat mempunyai sifat-sifat yang berikut:

Shear Modulus, $G = 2.4 \text{ MN/m}^2$

Modulus ricih, G = 2.4 MN/m²

Sample's height = 18 mm

Ketinggian sampel = 18 mm

Threshold energy for ozone crack growth, $G_z = 40 \text{ Jm}^{-2}$

Perambatan carikan akibat ozon, G_z = 40 Jm⁻²

Threshold energy for mechanical oxidative crack growth, $G_o = 10 \text{ kJm}^{-2}$

Perambatan carikan akibat oksidatif mekanikal, G_o = 10 kJm⁻²

Crack growth constant, $B = 5.0 \times 10^{-8} \text{ mcycle}^{-1}/\text{kJm}^{-2}$

Perambatan carikan konstan, B = 5.0 × 10⁻⁸ mcycle⁻¹/kJm⁻²

Constant β = 2

Konstan β = 2

- (i). Calculate the threshold cyclic strain for the bonded shear unit under linear shear stress-strain for ozone and mechanical oxidative cracking.

Kirakan kitaran terikan ambang bagi unit getah terikat dibawah kelakuan tegasan-terikan rincih linear bagi carikan ozon dan oksidatif mekanikal.

(8 marks/markah)

- (ii). Predict the crack growth per year if the maximum cyclic strain of 15% is applied 10^6 times per year.

Ramalkan pembentukan perambatan carik setahun jika kitaran terikan maksimum sebanyak 15% dikenakan 10^6 kali dalam masa setahun.

(7 marks/markah)

- (2). (a). Pneumatic conveying systems as shown in Figure 1 is exposed to mechanical fatigue cracks and generates noise due to unstable fluid flow during service. Pipes can also ‘breakout noise’ from machinery attached to the end of the pipe. The problem is often compounded by rigid fixing of the pipes to walls and other structures, which can amplify the vibrations. Discuss three (3) approaches to control the vibration and noise and suggests the most ergonomic method to reduce the vibration and increase the stability of the system

Sistem penghantar pneumatik seperti yang ditunjukkan dalam Rajah 1 terdedah kepada retakan kelesuan mekanikal dan menghasilkan bunyi bising akibat aliran bendalir yang tidak stabil

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semasa servis. Paip juga boleh menghasilkan bunyi bising daripada mesin yang dipasang pada hujung paip. Masalah ini ditambah dengan pemasangan paip yang tetap pada dinding dan struktur lain, yang boleh menguatkan getaran. Bincangkan tiga (3) pendekatan untuk mengawal getaran dan bunyi dan cadangkan kaedah yang paling ergonomik untuk mengurangkan getaran dan meningkatkan kestabilan system.

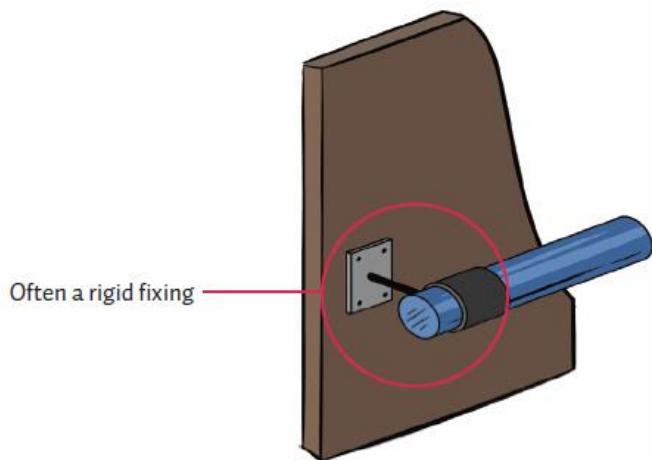


Figure 1 : Pneumatic conveying system

Rajah 1 : Sistem penghantar pneumatik

(12 marks/markah)

- (b). Briefly describe two commonly used rubber fender systems and compare the advantages and disadvantages of the rubber fenders to absorb impact forces during collision between fenders and vessel.

Terangkan secara ringkas dua sistem fender getah yang biasa digunakan dan bandingkan kelebihan dan kekurangan fender getah untuk menyerap daya impak semasa perlanggaran antara fender dan kapal.

(7 marks/markah)

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- (c). Explain functions of tire as a part of automobile and discuss three different frictional forces generated during driving on smooth, rough and sharpened road surface.

Terangkan fungsi tayar sebagai sebahagian daripada bahagian kereta dan bincangkan tiga daya geseran yang berbeza yang dijana semasa memandu di permukaan jalan yang licin, kasar dan tajam

(6 marks/markah)

PART B / BAHAGIAN B

- (3). A rectangular rubber block bearing, measuring of 150 mm x 75 mm x 16 mm (length x width x thickness), have the following rubber properties:

Satu segiempat tepat blok galas getah dengan ukuran 150 mm x 75 mm x 16 mm (panjang x lebar x tebal), mempunyai sifat-sifat getah seperti berikut:

Young's modulus = 3.25 MN/m²

Modulus Young = 3.25 MN/m²

Shear modulus = 0.81 MN/m²

Modulus ricih = 0.81 MN/m²

Bulk modulus = 1090 MN/m²

Modulus pukal = 1090 MN/m²

Correction factor = 0.64

Faktor pembetulan = 0.64

- (a). Assess the factors that contribute to the deviation of force-deformation behavior of the rubber bearing based on a statistical approach of rubber bearing.

Nilaikan faktor yang menyumbang kepada sisihan tingkah laku ubah bentuk daya galas getah berdasarkan pendekatan statistik galas getah.

(8 marks/markah)

- (b). If the rubber bearing is divided into 8 equal thickness sections by rigid shims, calculate the compression spring rate, shear spring rate and the ratio of K_c/K_s of the laminated rubber bearing.

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Jika galas getah ini dibahagikan kepada 8 bahagian yang mempunyai ketebalan yang sama oleh kepipis tegar, kirakan kadar mampatan spring, kadar ricihan spring dan nisbah K_c/K_s untuk galas rubber ini.

(12 marks/markah)

- (c). Justify the importance of lamination process in this rubber bearing design considerations.

Berikan justifikasi kepentingan proses laminasi dalam pertimbangan reka bentuk galas getah ini.

(5 marks/markah)

- (4). (a). A circular disc rubber bearing with measuring diameter of 600 mm and thickness 20 mm have the following rubber properties:

Satu disk bulat galas getah dengan ukuran diameter 600 mm dan ketebalan 18 mm diperbuat daripada getah yang mempunyai sifat-sifat getah seperti berikut:

Young's modulus = 3.34 MN/m²

Modulus Young = 3.34 MN/m²

Shear modulus = 1.05 MN/m²

Modulus ricih = 1.05 MN/m²

Bulk modulus = 1124 MN/m²

Modulus pukal = 1124 MN/m²

Correction factor = 0.67

Faktor pembetulan = 0.67

- (a). Propose a suitable approach to calculate the force deformation behaviour of the circular disk.

Cadangkan pendekatan yang sesuai untuk mengira kelakuan ubah bentuk daya cakera bulat.

(8 marks/markah)

- (b). If the rubber bearing is divided into 9 equal thickness sections by rigid shims, calculate the compression spring rate, shear spring rate and the ratio of K_c/K_s of the laminated rubber bearing.

Jika galas getah ini dibahagikan kepada 9 bahagian yang mempunyai ketebalan yang sama oleh kepis tegar, kirakan kadar mampatan spring, kadar ricihan spring dan nisbah K_c/K_s untuk galas getah ini.

(12 marks/markah)

- (c). Discuss the influence of instability in designing rubber for bridge bearing application.

Bincangkan kesan ketidakstabilan dalam mereka bentuk getah untuk aplikasi galas jambatan.

(5 marks/markah)

PART C / BAHAGIAN C

- (5). (a). Explain viscoelastic properties of rubber materials using Kelvin Voight and Maxwell models.

Terangkan sifat viskoelastik bahan getah menggunakan model Kelvin Voight dan Maxwell

(5 marks/markah)

- (b). Figure 2 shows time dependence of four types of materials with different value of the damping ratio ζ . Discuss the damping behaviour of each material and suggest the most suitable candidate material for efficient damping.

Rajah 2 menunjukkan kebergantungan masa bagi empat jenis bahan dengan nilai nisbah redaman ζ yang berbeza. Bincangkan tingkah laku redaman setiap bahan dan cadangkan calon bahan yang paling sesuai untuk redaman yang efisyen.

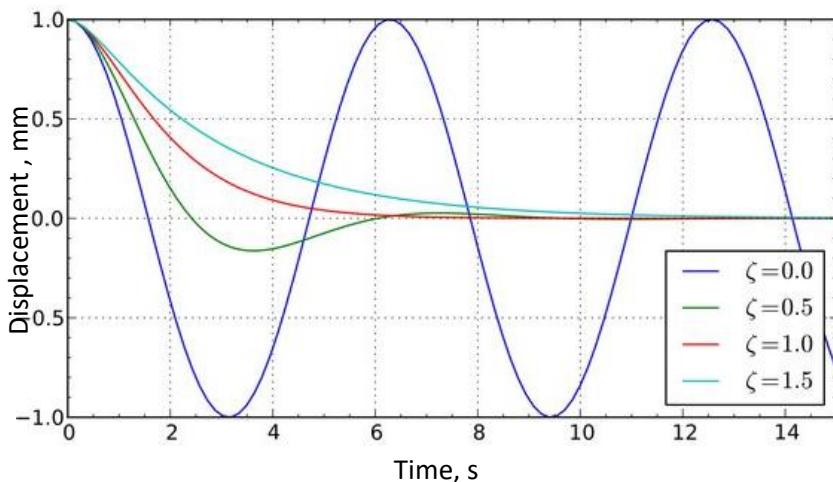


Figure 2 : A plot of displacement versus time for four materials with different damping ratio

Rajah 2 : Plot anjakan melawan masa masa untuk empat bahan dengan nisbah redaman yang berbeza

(10 marks/markah)

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- (c). A number of rubber engineering applications involve deformation of the product in a cyclic loading. Sketch idealized cyclic stress-strain curve of a vibration mounting on an engine moves repeatedly through a sinusoidal cyclic deformation. Identify the phase angle, δ and derive the complex modulus E^* and tangent of the phase angle δ ($\tan \delta$).

Beberapa aplikasi kejuruteraan getah melibatkan ubah bentuk produk semasa beban kitaran. Lakarkan kitaran lengkung tegasan-terikan ideal untuk cagak getaran yang bergerak melalui ubah bentuk kitaran gelombang sinus. Kenal pasti sudut fasa, δ dan terbitkan modulus kompleks E^ dan tangen daripada sudut fasa δ ($\tan \delta$).*

(10 marks/markah)

- (6). (a). Figure 3 shows vibration transmissibility (T_r) versus vibration frequency ratio (u) for a rubber mounting. Discuss anti-vibration effect at $u=0$, $u=1$, $u = 1.4142$ and $u > 1.4142$

Rajah 3 menunjukkan transmisibiliti (T_r) getaran berbanding nisbah frekuensi getaran (u) untuk cagak getah. Bincangkan kesan anti-getaran pada $u=0$, $u=1$, $u = 1.4142$ and $u > 1.4142$

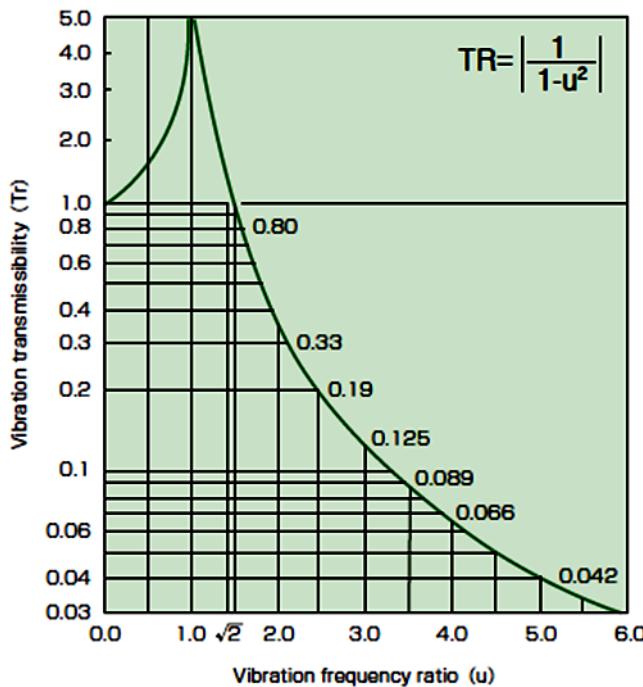


Figure 3: Transmissibility versus vibration frequency ratio for a rubber mounting

Rajah 3: Transmisibiliti melawan nisbah frekuensi getaran untuk satu cagak getah.

(10 marks/markah)

- (b). A machine is subjected to harmonic force of 66.3 mm amplitude distance that remains constant. The machine is connected to a rigid base with a rubber mounting that capable to reduce the amplitude distance by 42%. Calculate transmissibility of the anti-vibration rubber mounts.

Mesin dikenakan daya harmonik dengan jarak amplitud 66.3 mm yang tetap. Mesin disambungkan ke tapak rigid dengan cagak getah yang mampu mengurangkan jarak amplitud sebanyak 42%. Kira transmisibiliti cagak getah anti gegaran tersebut.

(5 marks/markah)

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- (c). Using appropriate damping models, derive viscous damping and interfacial damping mechanisms of a bridge bearing under shear deformation and suggest how to obtain total damping capacity of the materials.

Dengan menggunakan model redaman yang sesuai, terbitkan mekanisma redaman likat dan redaman antaramuka untuk galas jambatan di bawah ubah bentuk rincih dan cadangkan bagaimana untuk mendapatkan jumlah keseluruhan kapasiti redaman bahan tersebut.

(10 marks/markah)

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