
UNIVERSITI SAINS MALAYSIA

Second Semester Examination
Academic Session 2007/2008
*Peperiksaan Semester Kedua
Sidang Akademik 2007/2008*

April 2008
April 2008

**EMH 331/4 – *Finite Element Method
in Mechanical Engineering***
**Kaedah Unsur Terhingga
dalam Kejuruteraan Mekanik**

Duration : 3 hours
Masa : 3 jam

INSTRUCTIONS TO CANDIDATE:
ARAHAN KEPADA CALON :

Please check that this paper contains **EIGHT (8)** printed pages, **TWO (2)** pages appendix and **SIX (6)** questions before you begin the examination.

*Sila pastikan bahawa kertas soalan ini mengandungi **LAPAN (8)** mukasurat bercetak, **DUA (2)** mukasurat lampiran dan **ENAM (6)** soalan sebelum anda memulakan peperiksaan.*

Answer **FIVE (5)** questions.
*Jawab **LIMA (5)** soalan.*

Answer all questions in **English** or **Bahasa Malaysia** or a combination of both.
*Calon boleh menjawab semua soalan dalam **Bahasa Malaysia** atau **Bahasa Inggeris** atau kombinasi kedua-duanya.*

Start answering each question in a new page.
Setiap soalan mestilah dimulakan pada mukasurat yang baru.

Appendix/Lampiran:

1. The Given Selected Formula

[2 pages/mukasurat]

- Q1. [a] For the assemblage shown in Figure Q1[a], determine the nodal displacements, the forces in each element and the nodal reactions.

Untuk aturan yang ditunjukkan dalam Rajah S1[a], tentukan anjakan nod, daya dalam setiap elemen dan daya tindakbalas nod.

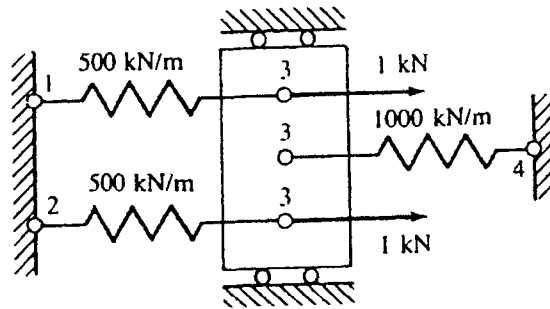


Figure Q1[a]
Rajah S1[a]

(50 marks/50 markah)

- [b] The triangular element is shown in Figure Q1[b]. The coordinates are given in units of millimeters. Assume plane stress conditions. Let Young's modulus $E = 210$ GPa, Poisson ratio, $\nu = 0.25$, and thickness $t = 10$ mm.
- Evaluate the stiffness matrix
 - If the nodal displacements are given as $u_1 = 0.02$ mm; $v_1 = 0.01$ mm; $u_2 = 0.05$ mm; $v_2 = 0.0$ mm; $u_3 = 0.03$ mm; $v_3 = 0.01$ mm, determine the element stresses σ_x , σ_y , τ_{xy} , σ_1 and σ_2 , and the principal angle θ_p .

Elemen segitiga adalah seperti yang ditunjukkan dalam Rajah S1[b]. Koordinat yang diberikan adalah dalam unit millimeter. Andaikan keadaan tegasan satah. Ambil modulus Young $E = 210$ GPa, nisbah Poisson, $\nu = 0.25$, dan ketebalan $t = 10$ mm.

- Kirakan matriks kekakuan
- Jika anjakan nod diberikan seperti berikut: $u_1 = 0.02$ mm; $v_1 = 0.01$ mm; $u_2 = 0.05$ mm; $v_2 = 0.0$ mm; $u_3 = 0.03$ mm; $v_3 = 0.01$ mm, kirakan tegasan elemen σ_x , σ_y , τ_{xy} , σ_1 and σ_2 , and sudut prinsipal θ_p .

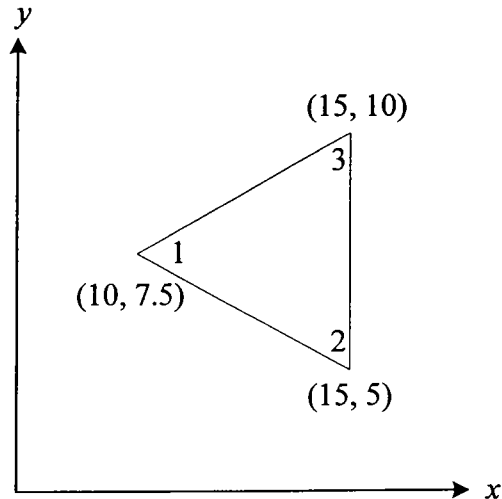


Figure Q1[b]
Rajah S1[b]

(50 marks/50 markah)

- Q2. [a] List and briefly describe the general steps in the finite element method.

Senaraikan dan terangkan dengan ringkas langkah-langkah umum dalam kaedah unsur terhingga.

(20 marks/20 markah)

- [b] Use the principle of minimum potential energy to find the equilibrium of displacement of spring system as shown in Figure Q2[b]. Briefly discuss your findings.

Gunakan kaedah prinsip minima tenaga potensi untuk menentukan keseimbangan anjakan pada pegas yang ditunjukkan dalam Rajah S2[b]. Terangkan dengan ringkas penemuan yang dicapai.

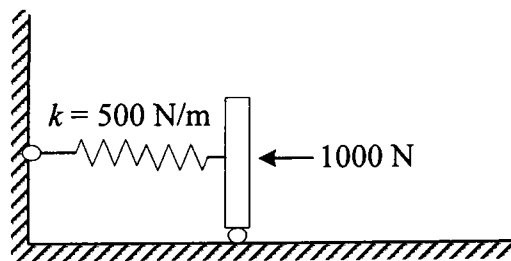


Figure Q2[b]
Rajah S2[b]

(50 marks/50 markah)

- [c] The bar shown in Figure Q2[c] has the following parameters: $E = 210 \text{ GPa}$, $A = 4.0 \times 10^{-4} \text{ m}^2$, and $L = 2 \text{ m}$, and the angle between x and \hat{x} is 50° . By assuming the global displacements that have been previously determined to be $d_{1x} = 0.3 \text{ mm}$, $d_{1y} = 0.0 \text{ mm}$ and $d_{2x} = 0.6 \text{ mm}$, $d_{2y} = 0.8 \text{ mm}$ respectively, determine the axial stress in the bar.

Bar yang ditunjukkan di dalam Rajah S2[c] mempunyai parameter-parameter berikut: $E = 210 \text{ GPa}$, $A = 4.0 \times 10^{-4} \text{ m}^2$, dan $L = 2 \text{ m}$, dan sudut di antara x dan untuk bar tersebut \hat{x} ialah 50° . Dengan mengandaikan anjakan global telah ditentukan sebelum ini iaitu $d_{1x} = 0.3 \text{ mm}$, $d_{1y} = 0.0 \text{ mm}$ dan $d_{2x} = 0.6 \text{ mm}$, $d_{2y} = 0.8 \text{ mm}$, tentukan tegasan paksi untuk bar tersebut.

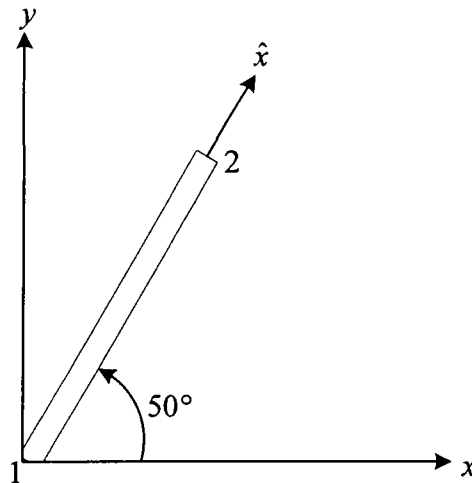


Figure Q2[c]
Rajah S2[c]

(30 marks/30 markah)

- Q3. [a] For the plane trusses shown in Figure Q3[a], determine the horizontal and vertical displacements of node 1 and the stresses in each element. All elements have $E = 210 \text{ GPa}$ and cross sectional area, $A = 4.0 \times 10^{-4} \text{ m}^2$.

Untuk kekuda satah yang ditunjukkan dalam Rajah S3[a], tentukan anjakan menegak dan melintang pada nod 1 dan tegasan di setiap elemen. Semua elemen mempunyai $E = 210 \text{ GPa}$ dan luas keratan rentas $A = 4.0 \times 10^{-4} \text{ m}^2$.

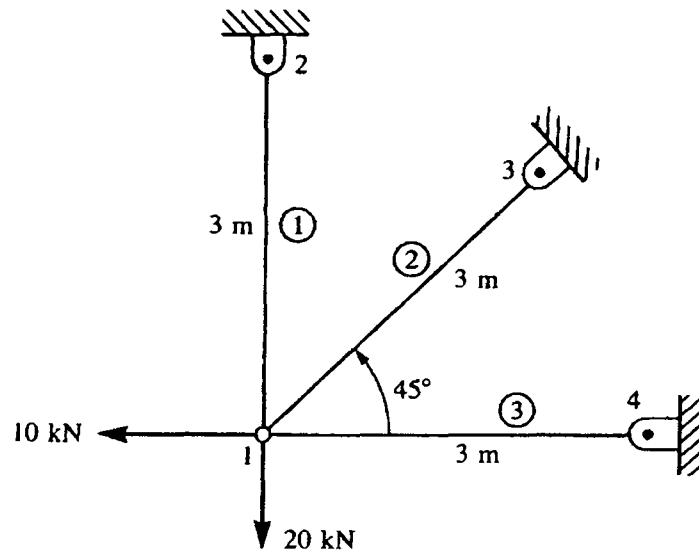


Figure Q3[a]
Rajah S3[a]

(70 marks/70 markah)

- [b] Derive the stiffness matrix, k for a linear elastic bar element by using Galerkin's method where the governing equation is given by

$$\frac{d}{dx} \left(AE \frac{du}{dx} \right) = 0. \quad A \text{ is cross-sectional area, } E \text{ is Young's modulus and } u \text{ is}$$

the displacement function.

Terbitkan matriks kekakuan, k untuk elemen bar linear elastik dengan menggunakan kaedah Galerkin di mana persamaan diberi oleh $\frac{d}{dx} \left(AE \frac{du}{dx} \right) = 0$. A ialah luas keratan rentas, E ialah modulus Young dan u ialah fungsi anjakan.

(30 marks/30 markah)

- Q4. [a] For the beam shown in Figure Q4[a], with the node numbers 1-2-3, determine

- (i) The displacements
- (ii) The slopes at the node
- (iii) The forces in each element
- (iv) The reactions

Untuk rasuk yang ditunjukkan dalam Rajah S4[a] dengan nombor-nombor nod 1-2-3, tentukan

- (i) Anjakan
- (ii) Kecerunan pada nod
- (iii) Daya pada setiap element
- (iv) Daya tindakbalas

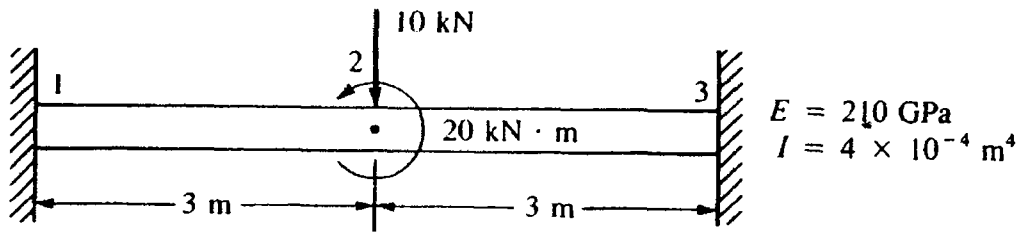


Figure Q4[a]
Rajah S4[a]

(60 marks/60 markah)

- [b] Consider the following displacement function for the two-noded bar element:

$$u = a_1 + a_2x^2$$

Discuss whether this displacement function is a valid or not valid displacement function. State the reasons.

Fungsi anjakan untuk 2 nod bar elemen diberikan seperti berikut:

$$u = a_1 + a_2x^2$$

Bincangkan samada fungsi anjakan ini sah atau tidak. Nyatakan alasannya.
(40 marks/40 markah)

- Q5. [a] For the composite wall idealized by the one-dimensional model shown in Figure Q5[a], determine the interface temperature between elements 1 & 2, and 2 & 3. For element 1, let thermal conductivity $K_{xx} = 15 \text{ W/(m}\cdot\text{°C)}$; for element 2, $K_{xx} = 10 \text{ W/(m}\cdot\text{°C)}$; and for element 3, $K_{xx} = 25 \text{ W/(m}\cdot\text{°C)}$. The left end has a constant temperature of 200°C and the right end has a constant temperature of 600°C .

Untuk dinding komposit yang diambil sebagai model satu dimensi seperti yang ditunjukkan dalam Rajah 5(a), tentukan suhu diantara permukaan elemen 1 & 2, dan 2 & 3. Untuk elemen 1, ambil aliran terma, $K_{xx} = 15 \text{ W/(m}\cdot\text{°C)}$; untuk elemen 2, $K_{xx} = 10 \text{ W/(m}\cdot\text{°C)}$; dan untuk elemen 3, $K_{xx} = 25 \text{ W/(m}\cdot\text{°C)}$. Suhu tetap dihujung kiri ialah 200°C manakala suhu ditetapkan dihujung kanan ialah 600°C .

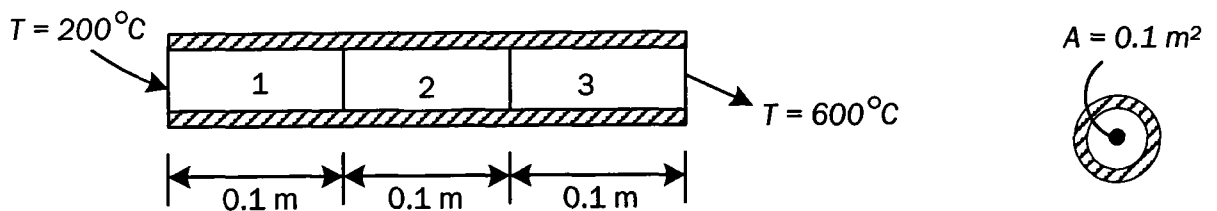


Figure Q5[a]
Rajah S5[a]

(40 marks/40 markah)

- [b] Calculate the stiffness matrix k and the force matrix f for the element shown in Figure Q5[b]. The conductivities are $K_{xx} = K_{yy} = 15 \text{ W/(m}\cdot\text{°C)}$ and the convection coefficient is $h = 20 \text{ W/(m}^2\cdot\text{°C)}$. Convection occurs across the i - m surface. The free-stream temperature is $T_\infty = 20^\circ\text{C}$. The coordinate are shown expressed in units of meters. Let the line source be $Q^* = 75 \text{ W/m}$ as located in the figure. Take the thickness of the element to be 1 m.

Untuk elemen segitiga seperti yang ditunjukkan dalam Rajah S5[b], tentukan matriks kekakuan k dan matriks daya f . Kekonduksian ialah $k_{xx} = k_{yy} = 15 \text{ W/(m}\cdot\text{°C)}$ dan pemalar perolakan ialah $h = 20 \text{ W/(m}^2\cdot\text{°C)}$. Perolakan berlaku pada permukaan nod i - m . Suhu sekeliling ialah $T_\infty = 20^\circ\text{C}$. Sumber haba Q^* ialah sumber garisan $Q^* = 75 \text{ W/m}$ dilokasi yang ditunjukkan dalam rajah. Koordinat ditunjukkan dalam unit m. Ambil ketebalan elemen 1 m.

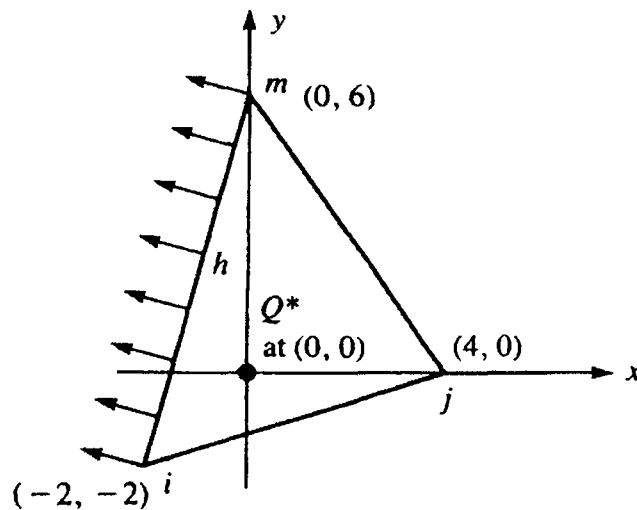


Figure Q5[b]
Rajah S5[b]

(60 marks/60 markah)

- Q6. [a] For the smooth pipe shown discretized in Figure Q6[a] with uniform cross section of 1 cm^2 , determine the flow velocities at the center and right end, knowing the velocity at the left end is $v_x = 2 \text{ cm/s}$.

Untuk satu paip licin yang dibahagikan seperti yang ditunjukkan dalam Rajah S6[a] dengan keratan rentas seragam 1 cm^2 , tentukan had laju aliran pada pusat dan hujung sebelah kanan, dengan mengetahui had laju pada sebelah kiri ialah $v_x = 2 \text{ cm/s}$.

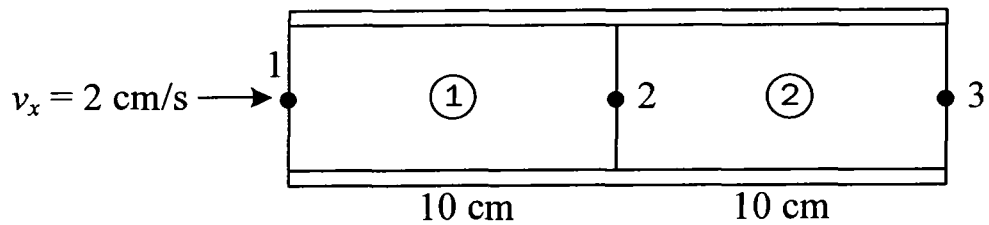


Figure Q6[a]
Rajah S6[a]

(30 marks/30 markah)

- [b] For the plane truss shown in Figure Q6[b], determine the displacements at node 1 and the axial stresses in each bar. Bar 1 is subjected to a temperature rise of 30°C . Let $E = 210 \text{ GPa}$, coefficient of thermal expansion $\alpha = 12 \times 10^{-6} (\text{mm/mm})/^{\circ}\text{C}$, and cross sectional area $A = 1 \times 10^{-2} \text{ m}^2$ for both bar elements.

Untuk kekuda satah yang ditunjukkan dalam Rajah S6[b], tentukan anjakan pada nod 1 dan tegasan paksi dalam setiap bar. Bar elemen 1 dikenakan suhu meningkat 30°C . Ambil modulus Young $E = 210 \text{ GPa}$, pekali pengembangan terma $\alpha = 12 \times 10^{-6} (\text{mm/mm})/^{\circ}\text{C}$ dan luas keratan rentas $A = 1 \times 10^{-2} \text{ m}^2$ untuk kedua dua bar elemen.

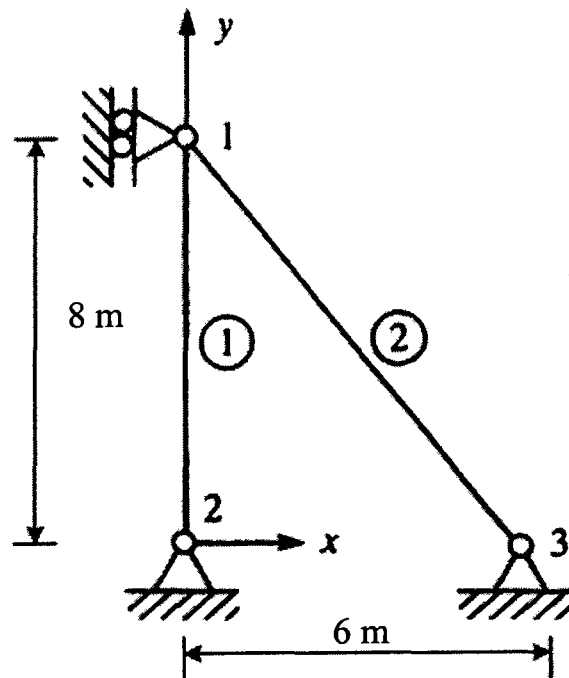


Figure Q6[b]
Rajah S6[b]

(70 marks/70 markah)

The given selected formula

Bar and Truss:

$$\begin{Bmatrix} \hat{d}_x \\ \hat{d}_y \end{Bmatrix} = \begin{bmatrix} C & S \\ -S & C \end{bmatrix} \begin{Bmatrix} d_x \\ d_y \end{Bmatrix} \quad \hat{d} = Td \quad T = \begin{bmatrix} C & S & 0 & 0 \\ -S & C & 0 & 0 \\ 0 & 0 & C & S \\ 0 & 0 & -S & C \end{bmatrix}$$

$$\underline{k} = \frac{AE}{L} \begin{bmatrix} C^2 & CS & -C^2 & -CS \\ & S^2 & -CS & -S^2 \\ & & C^2 & CS \\ \text{Symmetry} & & & S^2 \end{bmatrix} \quad \underline{\sigma} = \underline{C}' \underline{d} \quad \underline{C}' = \frac{E}{L} \begin{bmatrix} -1 & 1 \\ 1 & -1 \end{bmatrix} \begin{bmatrix} C & S & 0 & 0 \\ 0 & 0 & C & S \end{bmatrix}$$

Beam and Frame:

$$\hat{k} = \frac{EI}{L^3} \begin{bmatrix} 12 & 6L & -12 & 6L \\ 6L & 4L^2 & -6L & 2L^2 \\ -12 & -6L & 12 & -6L \\ 6L & 2L^2 & -6L & 4L^2 \end{bmatrix}$$

Plane Stress

$$\{\sigma\} = [D]\{\epsilon\} \quad [D] = \frac{E}{1-\nu^2} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & \frac{1-\nu}{2} \end{bmatrix}$$

$$[k] = tA[B]^T [D][B]$$

Plane Strain

$$\{\sigma\} = [D]\{\epsilon\} \quad [D] = \frac{E}{(1+\nu)(1-2\nu)} \begin{bmatrix} 1-\nu & \nu & 0 \\ \nu & 1-\nu & 0 \\ 0 & 0 & \frac{1-2\nu}{2} \end{bmatrix}$$

Triangular element strain

$$\{\epsilon\} = \frac{1}{2A} \begin{bmatrix} \beta_i & 0 & \beta_j & 0 & \beta_m & 0 \\ 0 & \gamma_i & 0 & \gamma_j & 0 & \gamma_m \\ \gamma_i & \beta_i & \gamma_j & \beta_j & \gamma_m & \beta_m \end{bmatrix} \begin{Bmatrix} u_i \\ v_i \\ u_j \\ v_j \\ u_m \\ v_m \end{Bmatrix}$$

$$\{\epsilon\} = \begin{bmatrix} B_i & B_j & B_m \end{bmatrix} \begin{Bmatrix} d_i \\ d_j \\ d_m \end{Bmatrix}$$

$$[B_i] = \frac{1}{2A} \begin{bmatrix} \beta_i & 0 \\ 0 & \gamma_i \\ \gamma_i & \beta_i \end{bmatrix} \quad [B_j] = \frac{1}{2A} \begin{bmatrix} \beta_j & 0 \\ 0 & \gamma_j \\ \gamma_j & \beta_j \end{bmatrix} \quad [B_m] = \frac{1}{2A} \begin{bmatrix} \beta_m & 0 \\ 0 & \gamma_m \\ \gamma_m & \beta_m \end{bmatrix}$$

Characteristics of triangular element

$$2A = \begin{bmatrix} 1 & x_i & y_i \\ 1 & x_j & y_j \\ 1 & x_m & y_m \end{bmatrix} \quad 2A = x_i(y_j - y_m) + x_j(y_m - y_i) + x_m(y_i - y_j)$$

$$\begin{aligned} \alpha_i &= x_j y_m - y_j x_m & \alpha_j &= y_i x_m - x_i y_m & \alpha_m &= x_i y_j - y_i x_j \\ \beta_i &= y_j - y_m & \beta_j &= y_m - y_i & \beta_m &= y_i - y_j \\ \gamma_i &= x_m - x_j & \gamma_j &= x_i - x_m & \gamma_m &= x_j - x_i \end{aligned}$$

x displacement:

$$u(x, y) = \frac{1}{2A} \{ (\alpha_i + \beta_i x + \gamma_i y) u_i + (\alpha_j + \beta_j x + \gamma_j y) u_j + (\alpha_m + \beta_m x + \gamma_m y) u_m \}$$

y displacement:

$$v(x, y) = \frac{1}{2A} \{ (\alpha_i + \beta_i x + \gamma_i y) v_i + (\alpha_j + \beta_j x + \gamma_j y) v_j + (\alpha_m + \beta_m x + \gamma_m y) v_m \}$$

$$N_i = \frac{1}{2A} (\alpha_i + \beta_i x + \gamma_i y)$$

$$N_j = \frac{1}{2A} (\alpha_j + \beta_j x + \gamma_j y)$$

$$N_m = \frac{1}{2A} (\alpha_m + \beta_m x + \gamma_m y)$$