

UNIVERSITI SAINS MALAYSIA

Peperiksaan Semester Pertama
Sidang Akademik 2005/2006

November 2005

EEM 223 - TERMOBENDALIR

Masa : 3 Jam

ARAHAN KEPADA CALON:-

Sila pastikan kertas peperiksaan ini mengandungi **SEMBILAN (9)** muka surat beserta **Lampiran (6 muka surat)** bercetak dan **ENAM (6)** soalan sebelum anda memulakan peperiksaan ini.

Jawab **LIMA (5)** soalan.

Agihan markah diberikan di sudut sebelah kanan soalan berkenaan.

Semua soalan hendaklah dijawab dalam Bahasa Malaysia.

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1. [a] Proses isoterma ($1 \rightarrow 2$) bagi gas sempurna diberi mengikut hukum ($pV = \text{malar}$). Tunjukkan kerja terlaku oleh gas adalah

An isothermal process ($1 \rightarrow 2$) for an ideal gas is given by the law ($pV = \text{constant}$). Show that the work done by the gas is

$$W = RT_1 \ln \frac{P_1}{P_2}$$

di sini R = pemalar gas
gas constant

P = tekanan
pressure

T = suhu
temperature

v = isipadu tentu
specific volume

(40 markah)

- [b] Udara dengan jisim 6kg mengembang secara boleh-balik daripada tekanan 13bar kepada 1.3bar mengikut hukum $pV^{1.2} = \text{malar}$. Jika isipadu udara pada keadaan awal ialah 0.1m^3 , tentukan:

Air with a mass of 6kg expands reversibly from a pressure of 13bar to 1.3bar using the law of $pV^{1.2} = \text{constant}$. If the initial volume of air is 0.1m^3 , determine:

- (i) kerja terlaku
work done
- (ii) pemindahan haba
heat transfer

(60 markah)

2. [a] Dengan menggunakan hukum pertama termodinamik bagi sistem tertutup (abaikan perubahan tenaga kinetik dan tenaga keupayaan), terbitkan perhubungan;

Using the first law of thermodynamics for a closed system (neglect the changes in the kinetic and potential energies), derive the relation:

$$Tds = dh - vdp$$

Di sini T = suhu, s = entropi, h = entalpi, v = isipadu tentu dan p = tekanan

where T = temperature, s = entropy, h = enthalpy, v = specific volume and p = pressure

(40 markah)

- [b] Pemanas air suapan beroperasi pada keadaan mantap mempunyai dua salur masuk dan satu salur keluar. Pada salur masuk yang pertama, wap masuk pada tekanan $p_1 = 700\text{kPa}$, $T_1 = 200^\circ\text{C}$ dengan kadar aliran jisim 40kg/s . Di salur masuk kedua, air cecair pada tekanan $p_2 = 700\text{kPa}$, $T_2 = 40^\circ\text{C}$ masuk dengan keluasan $A_2 = 25\text{cm}^2$. Cecair tepu pada tekanan $p_3 = 700\text{kPa}$ keluar dengan kadar dimana isipadu $0.06\text{m}^3/\text{s}$. Tentukan kadar aliran jisim di salur masuk kedua dan salur keluar, dan kirakan halaju di salur masuk kedua.

A feed water heater operating at steady state has two inlets and one exit. At the first inlet, water vapor enters at $p_1 = 700\text{kPa}$, $T_1 = 200^\circ\text{C}$ with a mass flow rate of 40kg/s . At the second inlet, liquid water at $p_2 = 700\text{kPa}$, $T_2 = 40^\circ\text{C}$ enters through an area of $A_2 = 25\text{cm}^2$. Saturated liquid at $p_3 = 700\text{kPa}$ exits with a volumetric flow rate $0.06\text{m}^3/\text{s}$. Determine the mass flow rate at the second inlet and at the exit, and the velocity at the second inlet.

(60 markah)

3. [a] Sebuah enjin mempunyai kecekapan 100%. Terangkan samada enjin tersebut melanggar hukum pertama termodinamik dan hukum kedua termodinamik. Nyatakan hukum kedua termodinamik berdasarkan ungkapan Clausius.

An engine has an efficiency of 100%. Explain whether or not it violates the first law and the second law of thermodynamics. State the second law of thermodynamics as expressed by Clausius.

(30 markah)

- [b] Wap memasuki sebuah turbin dengan tekanan 30bar, suhu 400°C , dan halaju 160m/s . Wap tepu pada 100°C keluar dengan halaju 100m/s . Pada keadaan mantap, turbin menghasilkan kerja bersamaan 540kW setelah wap mengalir melalui turbin. Haba terpindah daripada turbin ke persekitaran adalah 10kW . Abaikan tenaga keupayaan dan

Steam enters a turbine with a pressure of 30bar, a temperature 400°C , and a velocity of 160m/s . Saturated vapor at 100°C exits with a velocity of 100m/s . At steady state, the turbine develops work equal to 540kW of steam flowing through the turbine. Heat transfer from the turbine to its surroundings is 10kW . Neglect the potential energy and

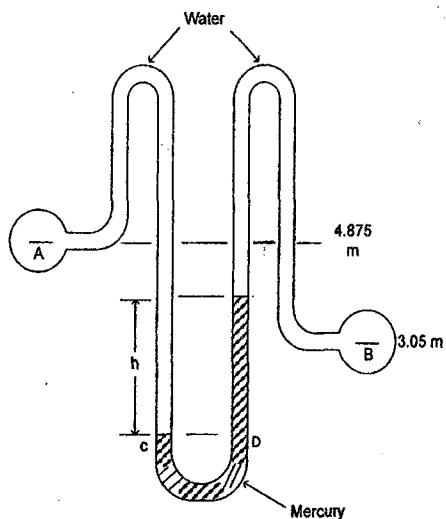
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- (i) tentukan kadar aliran jisim wap
determine the mass flow rate of steam
- (ii) tunjukkan proses adalah tak boleh balik
show that the process is irreversible
- (iii) kirakan perubahan entropi
calculate the change of entropy

(70 markah)

4. [a] Kebuk A dan B mengandungi air masing-masing pada tekanan 275kPa dan 140kPa seperti yang ditunjukkan dalam Rajah S4[a]. Apakah pesongan raksa di dalam tolok perbezaan.

Vessels A and B contain water pressure of 275kPa and 140kPa respectively as shown in Figure Q4[a]. What is the deflection of the mercury in the differential gauge?



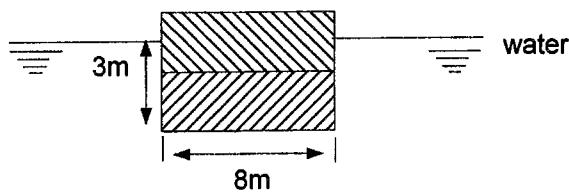
Rajah S4[a]
Figure Q4[a]

(30 markah)

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- [b] Tongkang seperti yang ditunjukkan dalam Rajah S4[b] dibebankan supaya pusat graviti tongkang dan beban pada paras permukaan air. Adakah tongkang itu stabil?

The barge shown in Figure Q4[b] is loaded such that the center of gravity of the barge and the load is at the waterline. Is the barge stable?



Rajah S4[b]
Figure Q4[b]

(30 markah)

- [c] Pelumba seretan meletakkan kole di atas sebuah dulang mengufuk sedangkan dia memecut pada kadar 7m/s^2 seperti yang ditunjukkan dalam Rajah S4[c]. Kole mempunyai kedalaman 10cm dan bergaris pusat 6cm dan mengandungi kopi kedalaman 7cm pada keadaan rehat.

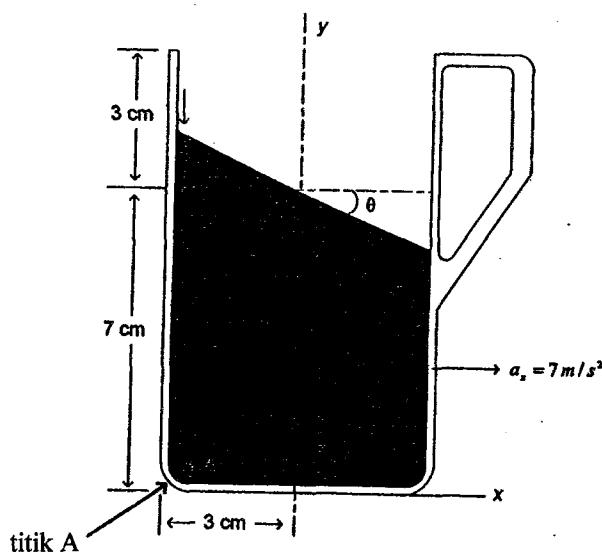
A drag racer rests her coffee mug on a horizontal tray while she accelerates at 7m/s^2 as shown in Figure Q4[c]. The mug is 10cm deep and 6cm in diameter and contains coffee 7cm deep at rest.

- (i) Anggapkan pecutan jasad tegar bagi kopi, tentukan samada kopi tertumpah keluar daripada kole.

Assuming rigid body acceleration of the coffee, determine whether it will spill out the mug.

- (ii) Kirakan tekanan tolak pada bucu di titik A jika ketumpatan kopi ialah 1010 kg/m^3 .

Calculate the gauge pressure in the corner at point A if the density of coffee is 1010 kg/m^3 .



Rajah S4[c]
Figure Q4[c]

(40 markah)

5. [a] Terangkan dengan bantuan gambarajah yang kemas, susunan sebuah meter venturi dan terangkan tujuan kegunaan setiap susunan.

Describe, with the help of a neat diagram, the arrangement of a venturi meter and explain its mode of operation.

(30 markah)

- [b] Terbitkan persamaan Bernoulli bagi aliran bendalir tak boleh mampat tanpa geseran berdasarkan keabadian momentum.

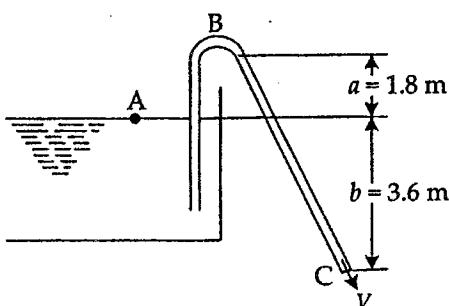
Derive Bernoulli's equation for the flow of an incompressible frictionless fluid from the conservation of momentum.

(30 markah)

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- [c] Sebuah sifon mempunyai lubang bergaris pusat 75mm dan terdiri daripada paip lengkok dengan lengkokan 1.8m di atas aras air meluah ke atmosfera pada aras 3.6m dibawah paras air (Rajah S5[c]). Tentukan halaju, luahan dan tekanan mutlak dilengkokan jika tekanan atmosfera adalah bersamaan 10m air. Abaikan kehilangan disebabkan geseran.

A siphon has a uniform circular bore of 75mm diameter and consists of a bent pipe with its crest 1.8m above water level discharging into the atmosphere at a level 3.6m below water level (Figure Q5[c]). Find the velocity, the discharge and the absolute pressure at crest level if the atmospheric pressure is equivalent to 10m of water. Neglect losses due to friction.



Rajah S5[c]
Figure Q5[c]

(40 markah)

6. [a] Bezakan perkara-perkara berikut dengan memberikan contoh:

Differentiate the following with examples:

- (i) Aliran mampat dan aliran tak mampat
Compressible and incompressible flows
- (ii) Bendalir Newton dan bukan Newton
Newtonian and non-Newtonian fluids
- (iii) Aliran mantap dan tak mantap
Steady and unsteady flow

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(iv) Aliran lamina dan gelora
Laminar and turbulent flow

(30 markah)

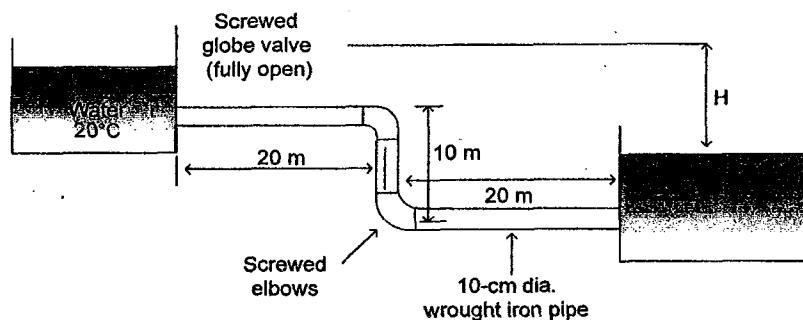
- [b] Apakah kehilangan geseran dan kehilangan kecil di dalam paip? Berdasarkan apakah faktor geseran bergantung bagi 'paip licin'? Berikan contoh-contoh paip licin.

What are friction and minor losses in pipes? On what does friction factor depend for 'smooth pipes'? Give examples of such pipes.

(20 markah)

- [c] Minyak akan dipindahkan daripada tangki dibahagian atas ke tangki dibahagian bawah pada kadar $0.03\text{m}^3/\text{s}$ seperti yang ditunjukkan dalam Rajah S6[c]. Tentukan aras H bagi tangki-tangki tersebut. (Ambil ketumpatan minyak sebagai 980kg/m^3 dan kelikatan dinamik sebagai 0.02Ns/m^2)

The oil will be transported from the upper reservoir to the lower reservoir at a rate of $0.03\text{m}^3/\text{s}$ as shown in Figure Q6[c]. Determine the difference in elevation H of the reservoirs. (Take the oil density as 980kg/m^3 and dynamic viscosity as 0.02Ns/m^2)



Rajah S6[c]
Figure Q6[c]

(50 markah)

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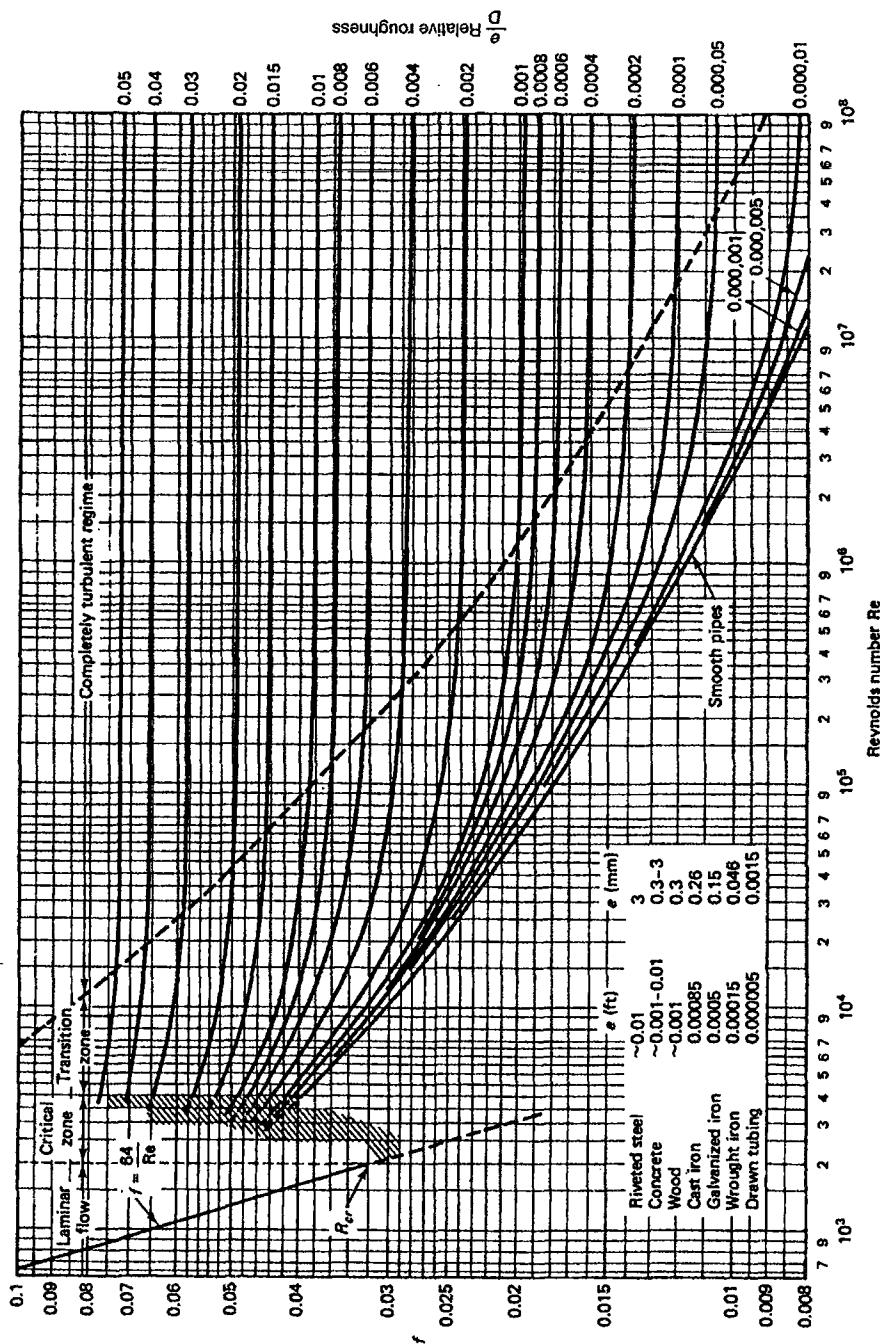


Figure 7.13 Moody diagram. (From L. F. Moody, *Trans. ASME*, Vol. 66, 1944.)

TABLE 7.2 Nominal Loss Coefficients K (Turbulent Flow)^a

Type of fitting	Screwed			Flanged			
	Diameter	1 in.	2 in.	4 in.	2 in.	4 in.	8 in.
Globe valve (fully open)		8.2	6.9	5.7	8.5	6.0	5.8
(half open)		20	17	14	21	15	14
(one-quarter open)		57	48	40	60	42	41
Angle valve (fully open)		4.7	2.0	1.0	2.4	2.0	2.0
Swing check valve (fully open)		2.9	2.1	2.0	2.0	2.0	2.0
Gate valve (fully open)		0.24	0.16	0.11	0.35	0.16	0.07
Return bend		1.5	.95	.64	0.35	0.30	0.25
Tee (branch)		1.8	1.4	1.1	0.80	0.64	0.58
Tee (line)		0.9	0.9	0.9	0.19	0.14	0.10
Standard elbow		1.5	0.95	0.64	0.39	0.30	0.26
Long sweep elbow		0.72	0.41	0.23	0.30	0.19	0.15
45° elbow		0.32	0.30	0.29			
Square-edged entrance					0.5		
Reentrant entrance					0.8		
Well-rounded entrance					0.03		
Pipe exit					1.0		
	Area ratio						
Sudden contraction ^b		2:1			0.25		
		5:1			0.41		
		10:1			0.46		
	Area ratio A/A_0						
Orifice plate		1.5:1			0.85		
		2:1			3.4		
		4:1			29		
		$\geq 6:1$			$2.78 \left(\frac{A}{A_0} - 0.6 \right)^2$		
Sudden enlargement ^c					$\left(1 - \frac{A_1}{A_2} \right)^2$		
90° miter bend (without vanes)					1.1		
(with vanes)					0.2		
General contraction		(30° included angle) (70° included angle)			0.02 0.07		

^aValues for other geometries can be found in *Technical Paper 410*, The Crane Company, 1957.^bBased on exit velocity V_2 .^cBased on entrance velocity V_1 .

Table A.6 Superheated water

T °C	v m ³ /kg	u kJ/kg	h kJ/kg	s kJ/kg · K	v m ³ /kg	u kJ/kg	h kJ/kg	s kJ/kg · K	v m ³ /kg	u kJ/kg	h kJ/kg	s kJ/kg · K
<i>P = 0.01 MPa (45.81°C)</i>					<i>P = 0.05 MPa (81.33°C)</i>					<i>P = 0.10 MPa (99.63°C)</i>		
Sat. 50	14.674 14.869	2437.9 2443.9	2584.7 2592.6	8.1502 8.1749	3.240 3.418	2483.9 2511.6	2645.9 2682.5	7.5939 7.6947	1.6940 1.6958	2506.1 2506.7	2675.5 2676.2	7.3594 7.3614
100	17.196	2515.5	2687.5	8.4479	3.889	2585.6	2780.1	7.9401	1.9364	2582.8	2776.4	7.6134
150	19.512	2587.9	2783.0	8.6882	4.356	2659.9	2877.7	8.1580	2.172	2658.1	2875.3	7.8343
200	21.825	2661.3	2879.5	8.9038	4.820	2735.0	2976.0	8.3556	2.406	2733.7	2974.3	8.0333
250	24.136	2736.0	2977.3	9.1002	5.284	2811.3	3075.5	8.5373	2.639	2810.4	3074.3	8.2158
300	26.445	2812.1	3076.5	9.2813	5.729	2968.5	3279.6	8.8642	3.103	2967.9	3278.2	8.5435
400	31.063	2968.9	3279.6	9.6077	6.209	3279.6	3648.2	9.2103	3.565	3131.6	3488.1	8.8342
500	35.679	3132.3	3489.1	9.8978	7.134	3132.0	3488.7	9.1546	3.832	3131.6	3488.1	8.8342
600	40.295	3302.5	3705.4	10.1608	8.057	3302.2	3705.1	9.4178	4.028	3301.9	3704.4	9.0976
700	44.911	3479.6	3928.7	10.4028	8.981	3479.4	3928.5	9.6599	4.490	3479.2	3928.2	9.3398
800	49.526	3663.8	4159.0	10.5281	9.904	3663.6	4158.9	9.8852	4.952	3663.5	4158.6	9.5562
900	54.141	3855.0	4396.4	10.8396	10.828	3854.9	4396.3	10.0967	5.414	3854.8	4396.1	9.7767
1000	58.757	4053.0	4640.6	11.0393	11.751	4052.9	4640.5	10.2964	5.875	4052.8	4640.3	9.9764
1100	63.372	4257.5	4891.2	11.2287	12.674	4257.4	4891.1	10.4859	6.337	4257.3	4891.0	10.1659
1200	67.987	4467.9	5147.8	11.4091	13.597	4467.8	5147.7	10.6662	6.799	4467.7	5147.6	10.3463
1300	72.602	4683.7	5409.7	11.5811	14.521	4683.6	5409.6	10.8382	7.260	4683.5	5409.5	10.5183
<i>P = 0.20 MPa (120.23°C)</i>					<i>P = 0.30 MPa (133.55°C)</i>					<i>P = 0.40 MPa (143.63°C)</i>		
Sat.	0.8857 0.9596	2529.5 2576.9	2706.7 2768.8	7.1272 7.2795	0.6058 0.6339	2543.6 2570.8	2725.3 2761.0	6.9919 7.0778	0.4625 0.4708	2553.6 2564.5	2738.6 2752.8	6.8959 6.9299
150	1.0803	2654.4	2870.5	7.5065	0.7163	2650.7	2865.6	7.3115	0.5342	2646.8	2860.5	7.1706
200	1.1988	2731.2	2971.0	7.7086	0.7964	2728.7	2967.6	7.5166	0.5951	2726.1	2964.2	7.3789
250	1.3162	2808.6	3071.8	7.8926	0.8753	2806.7	3069.3	7.7022	0.6548	2804.8	3066.8	7.5662
300	1.5493	2966.7	3276.6	8.2218	1.0315	2965.6	3275.0	8.0330	0.7726	2964.4	3273.4	7.8985
400	1.7814	3130.8	3487.1	8.5133	1.1867	3130.0	3486.0	8.3251	0.8893	3129.2	3484.9	8.1913
500	2.013	3301.4	3704.0	8.7770	1.3414	3300.8	3703.2	8.5892	1.0055	3300.2	3702.4	8.4558
600	2.244	3478.8	3927.6	9.0194	1.4957	3478.4	3927.1	8.8319	1.1215	3477.9	3926.5	8.6987
700	2.475	3663.1	4158.2	9.2449	1.6499	3652.9	4157.8	9.0576	1.2372	3662.4	4157.3	8.9244
800	2.705	3854.5	4395.8	9.4566	1.8041	3854.2	4395.4	9.2692	1.3529	3853.9	4395.1	9.1362
900	2.937	4052.5	4640.6	9.6563	1.9581	4052.3	4639.7	9.4690	1.4685	4052.0	4639.4	9.3360
1000	3.168	4257.0	4890.7	9.8458	2.1121	4256.8	4890.4	9.6585	1.5840	4256.5	4890.2	9.5256
1100	3.399	4467.5	5147.5	10.0262	2.2661	4467.2	5147.1	9.8389	1.6996	4467.0	5146.8	9.7060
1200	3.630	4683.2	5409.3	10.1982	2.4201	4683.0	5409.0	10.0110	1.8151	4682.8	5408.8	9.8780
<i>P = 0.50 MPa (151.86°C)</i>					<i>P = 0.60 MPa (158.85°C)</i>					<i>P = 0.80 MPa (170.43°C)</i>		
Sat.	0.3749 0.4249	2561.2 2542.9	2748.7 2855.4	6.8213 7.0592	0.3157 0.3520	2567.4 2638.9	2756.8 2850.1	6.7600 6.9865	0.2404 0.2608	2576.8 2630.6	2769.1 2839.3	6.6828 6.8158
200	0.4744	2723.5	2960.7	7.2709	0.3938	2720.9	2957.2	7.1816	0.2931	2715.5	2950.0	7.0384
250	0.5226	2802.9	3064.2	7.4599	0.4344	2801.0	3061.6	7.3724	0.3241	2797.2	3056.5	7.2328
300	0.5701	2882.6	3167.7	7.6329	0.4742	2881.2	3165.7	7.5464	0.3544	2878.2	3161.7	7.4089
400	0.6173	2963.2	3271.9	7.7938	0.5137	2962.1	3270.3	7.7079	0.3843	2959.7	3267.1	7.5716
500	0.7109	3128.4	3483.9	8.0873	0.5920	3127.6	3482.8	8.0021	0.4433	3126.0	3480.6	7.8673
600	0.8041	3299.6	3701.7	8.3753	0.6697	3299.1	3700.9	8.2674	0.5018	3297.9	3699.4	8.1333
700	0.8969	3477.5	3925.9	8.5952	0.7472	3477.0	3925.3	8.5107	0.5601	3476.2	3924.2	8.3770
800	0.9896	3662.1	4156.9	8.8211	0.8245	3661.8	4156.5	8.7367	0.6181	3661.1	4155.6	8.6033
900	1.0822	3853.6	4394.7	9.0329	0.9017	3853.4	4394.4	8.9486	0.6761	3852.8	4393.7	8.8153
1000	1.1747	4051.8	4639.1	9.2328	0.9788	4051.5	4638.8	9.1485	0.7340	4051.0	4638.2	9.0153
1100	1.2672	4256.3	4889.9	9.4224	1.0559	4256.1	4889.6	9.3381	0.7919	4255.6	4889.1	9.2050
1200	1.3596	4466.8	5146.6	9.6029	1.1330	4466.5	5146.3	9.5185	0.8497	4466.1	5145.9	9.3855
1300	1.4521	4682.5	5408.6	9.7749	1.2101	4682.3	5408.3	9.6906	0.9076	4681.8	5407.9	9.5575

Table A.2 Ideal-gas specific heats of various common gases at 300 K

Gas	Formula	Gas constant, R kJ/kg · K	C_p kJ/kg · K	C_v kJ/kg · K	γ
Air	—	0.2870	1.005	0.718	1.400
Argon	Ar	0.2081	0.5203	0.3122	1.667
Butane	C_4H_{10}	0.1433	1.7164	1.5734	1.091
Carbon dioxide	CO_2	0.1889	0.846	0.657	1.289
Carbon monoxide	CO	0.2968	1.040	0.744	1.400
Ethane	C_2H_6	0.2765	1.7662	1.4897	1.186
Ethylene	C_2H_4	0.2964	1.5482	1.2518	1.237
Helium	He	2.0769	5.1926	3.1156	1.667
Hydrogen	H_2	4.1240	14.307	10.183	1.405
Methane	CH_4	0.5182	2.2537	1.7354	1.299
Neon	Ne	0.4119	1.0299	0.6179	1.667
Nitrogen	N_2	0.2968	1.039	0.743	1.400
Octane	C_8H_{18}	0.0729	1.7113	1.6385	1.044
Oxygen	O_2	0.2598	0.918	0.658	1.395
Propane	C_3H_8	0.1885	1.6794	1.4909	1.126
Steam	H_2O	0.4615	1.8723	1.4108	1.327

Reference:

Yunus A. Cengel and Michael A. Boles, Thermodynamics – An Engineering Approach, 4th Edition, McGraw-Hill, 2002.